

Verification of assumptions in dynamics of lattice structures

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Outline of presentation

1. Motivation
2. Examples of lattice structures
3. Joints in lattice structures
4. „Pin joint“ assumption
5. Real structural behavior in dynamics
6. Examples
7. Conclusions

1. Motivation

Methods in Structural Health Monitoring

1.1. Global methods

Vibration-based methods (Operational Modal Analysis)

$$M\ddot{\mathbf{q}} + D_0\dot{\mathbf{q}} + K\mathbf{q} = \mathbf{w}$$

$$\mathbf{y} = \mathbf{C}_q\mathbf{q} + \mathbf{C}_v\dot{\mathbf{q}} + \mathbf{C}_a\ddot{\mathbf{q}}$$

Transformation to state space $\mathbf{x} = \begin{bmatrix} \mathbf{q} \\ \dot{\mathbf{q}} \end{bmatrix}$

$$\begin{cases} \dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{w} \\ \mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}\mathbf{w} + \mathbf{v} \end{cases}$$

Identified mass, damping and stiffness matrices

Measured quantities

1.2. Local methods

Ultrasonic waves propagation, Vibrothermography

2. Examples of lattice structures

A large number of structures are build of system of rods (beams) commonly called trusses, frames and lattice structures.



Truss bridges

Transmission towers

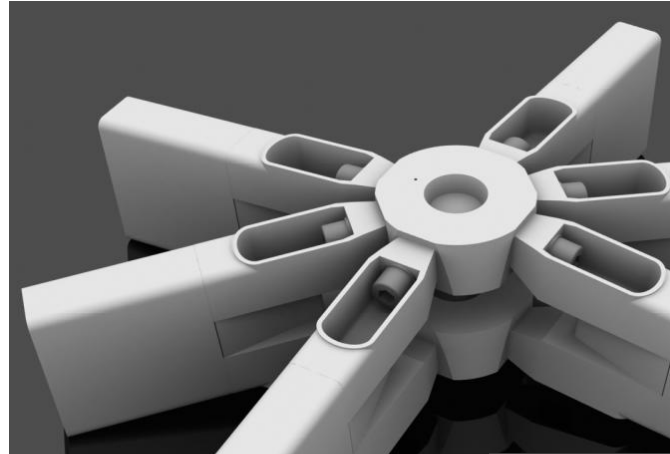


Double layer grids

3. Joints in lattice structures

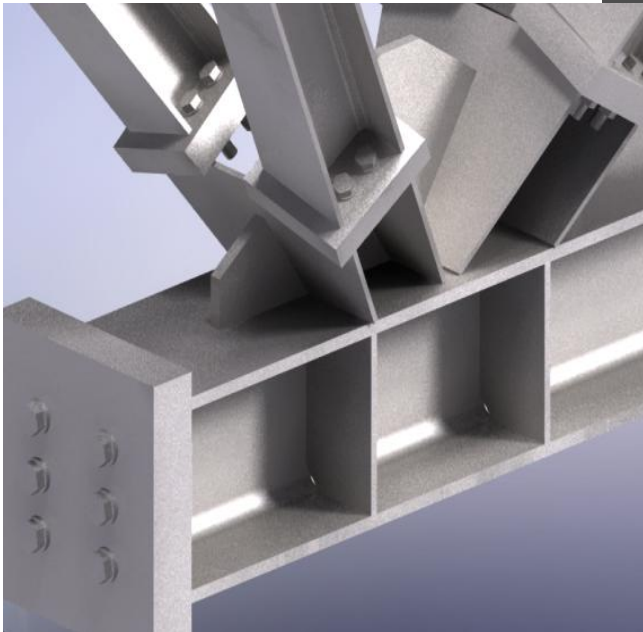
In most cases in such structures elements are rigidly connected.

*FF-System
Free form*

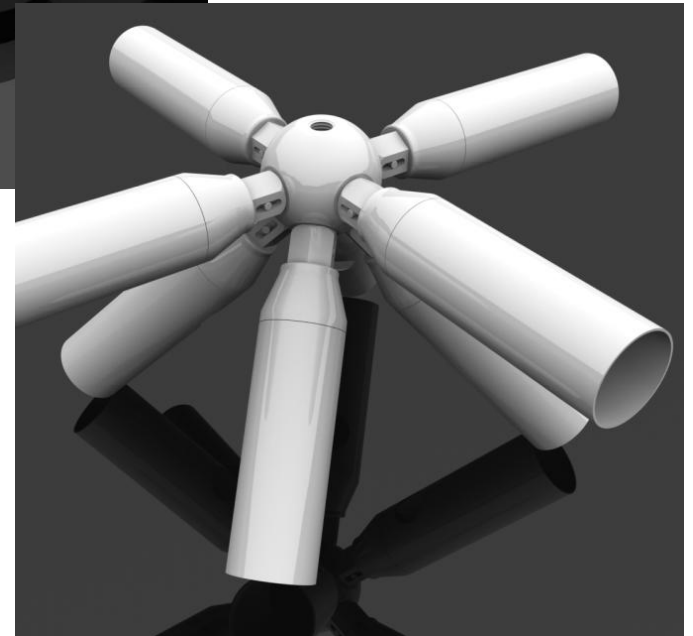


*Novum System
Products*

Classical welded joint

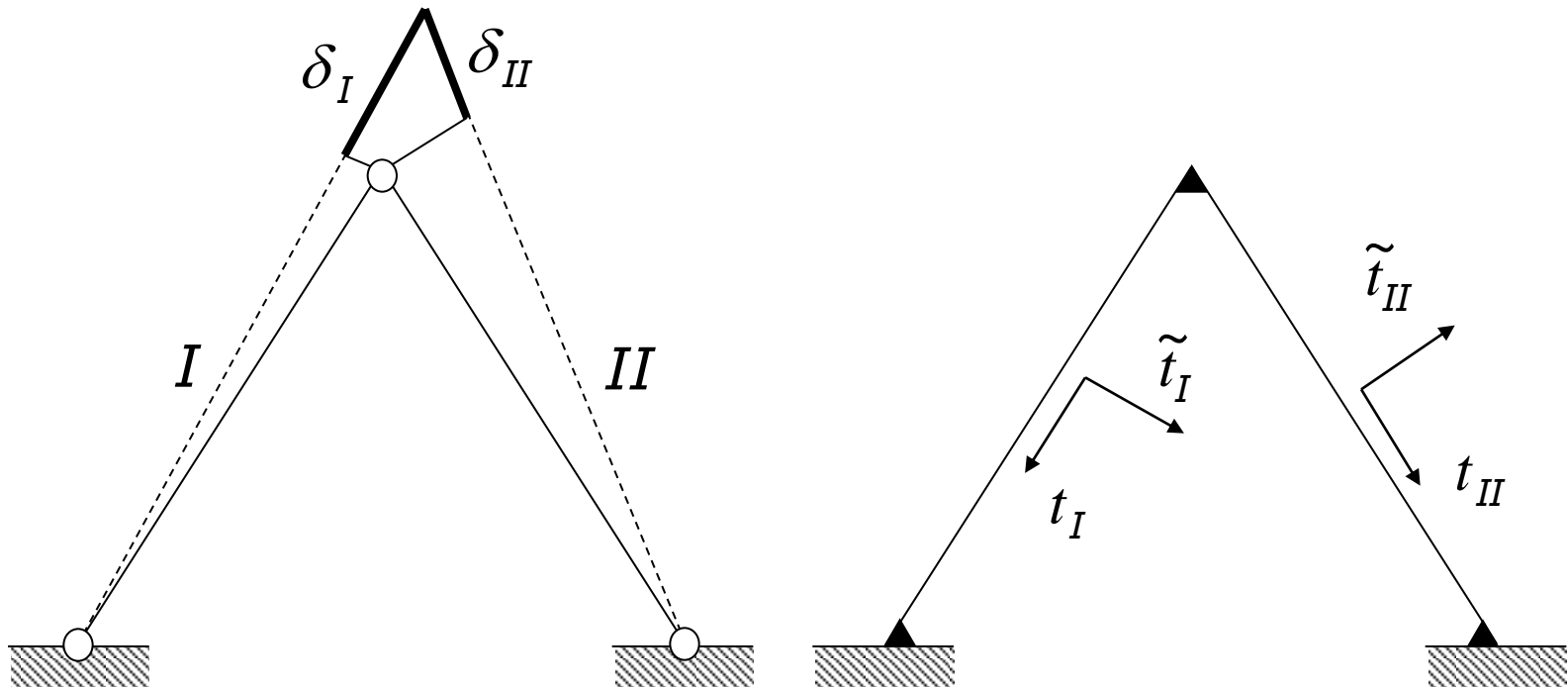


*KK-System
Kugel
knoten*



4. „Pin joint” assumption

The origin of the „pin joint” assumption



Equilibrium equations

- for pin joint

$$\begin{cases} C_I t_{I_1} \delta_I + C_{II} t_{II_1} \delta_{II} = P_1 \\ C_I t_{I_2} \delta_I + C_{II} t_{II_2} \delta_{II} = P_2 \end{cases} \quad C_I = \frac{EA_I}{l_I}$$

- for rigid connection

$$\begin{cases} -\tilde{K}_I \tilde{t}_{I_1} \tilde{\kappa}_I + C_I t_{I_1} \delta_I - \tilde{K}_{II} \tilde{t}_{II_1} \tilde{\kappa}_{II} + C_{II} t_{II_1} \delta_{II} = P_1 \\ -\tilde{K}_I \tilde{t}_{I_2} \tilde{\kappa}_I + C_I t_{I_2} \delta_I - \tilde{K}_{II} \tilde{t}_{II_2} \tilde{\kappa}_{II} + C_{II} t_{II_2} \delta_{II} = P_2 \\ \tilde{K}_I \vartheta_I + \tilde{K}_{II} \vartheta_{II} = 0 \end{cases} \quad \tilde{K}_I = \frac{EI_I}{l_I^3}$$

$$\frac{C}{\tilde{K}} = \frac{EA}{EI} l^2 = \left(\frac{l}{i} \right)^2 \rightarrow 10^4$$

Where

$$t_{I_1} \delta_I \text{ and } \tilde{t}_{I_1} \tilde{\vartheta}_I$$

are of the same order then the product

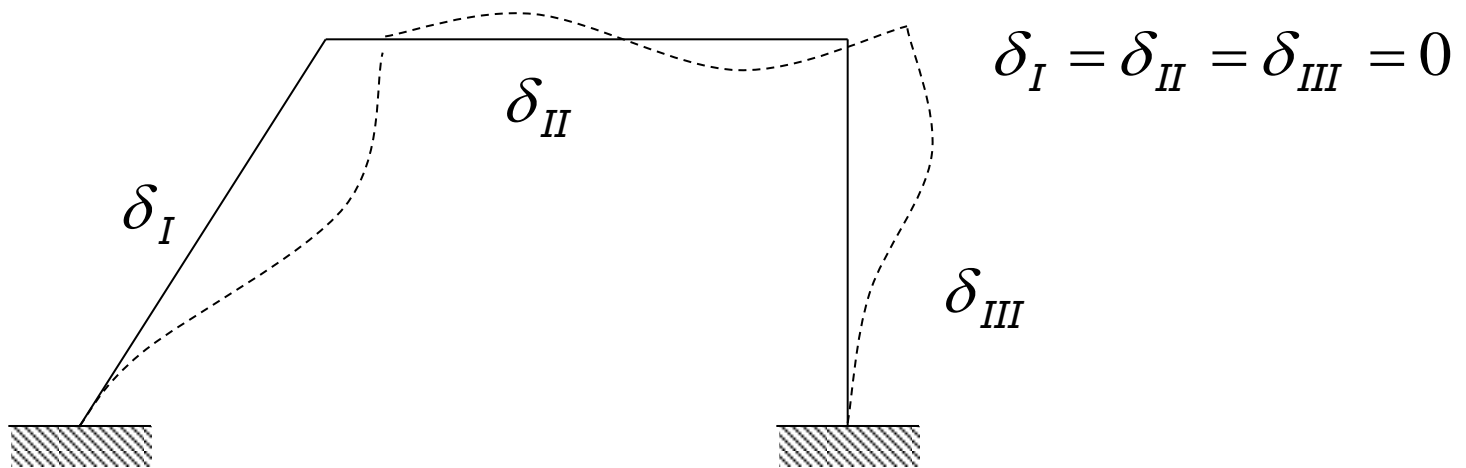
$$\tilde{K}_I \tilde{t}_{I_1} \tilde{\vartheta}_I$$

can be neglected comparing with $C_I t_{I_1} \delta_I$

$$\tilde{K}_I \tilde{t}_{I_1} \tilde{\vartheta}_I \lll C_I t_{I_1} \delta_I$$

Neglecting bending term is equivalent to the assumption that two elements are „pin joint“.

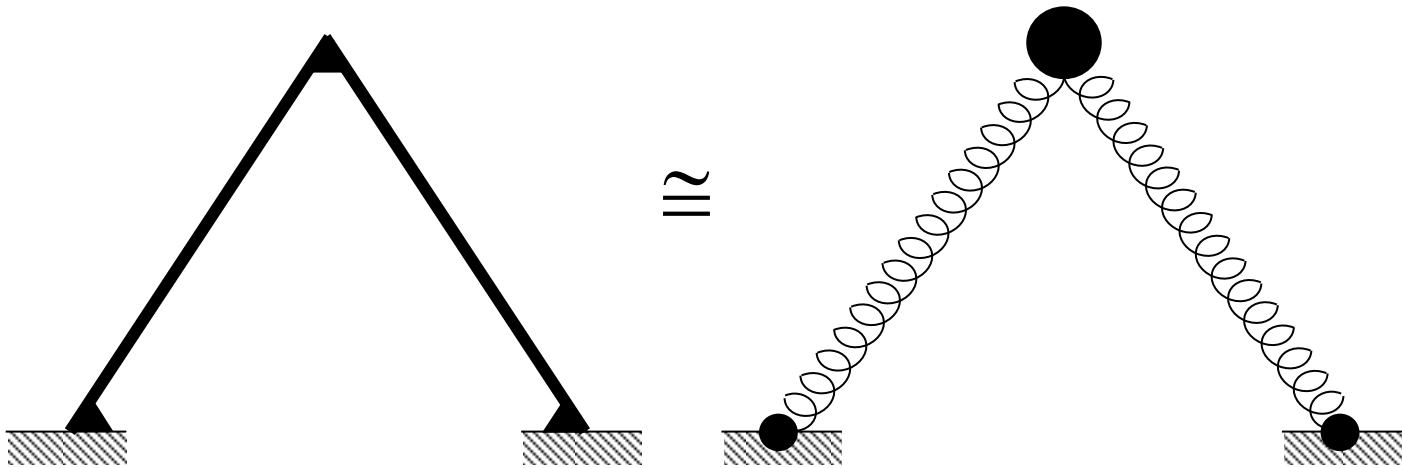
For structures when displacements are caused by bending only, „pin joint” assumption can not be applied.



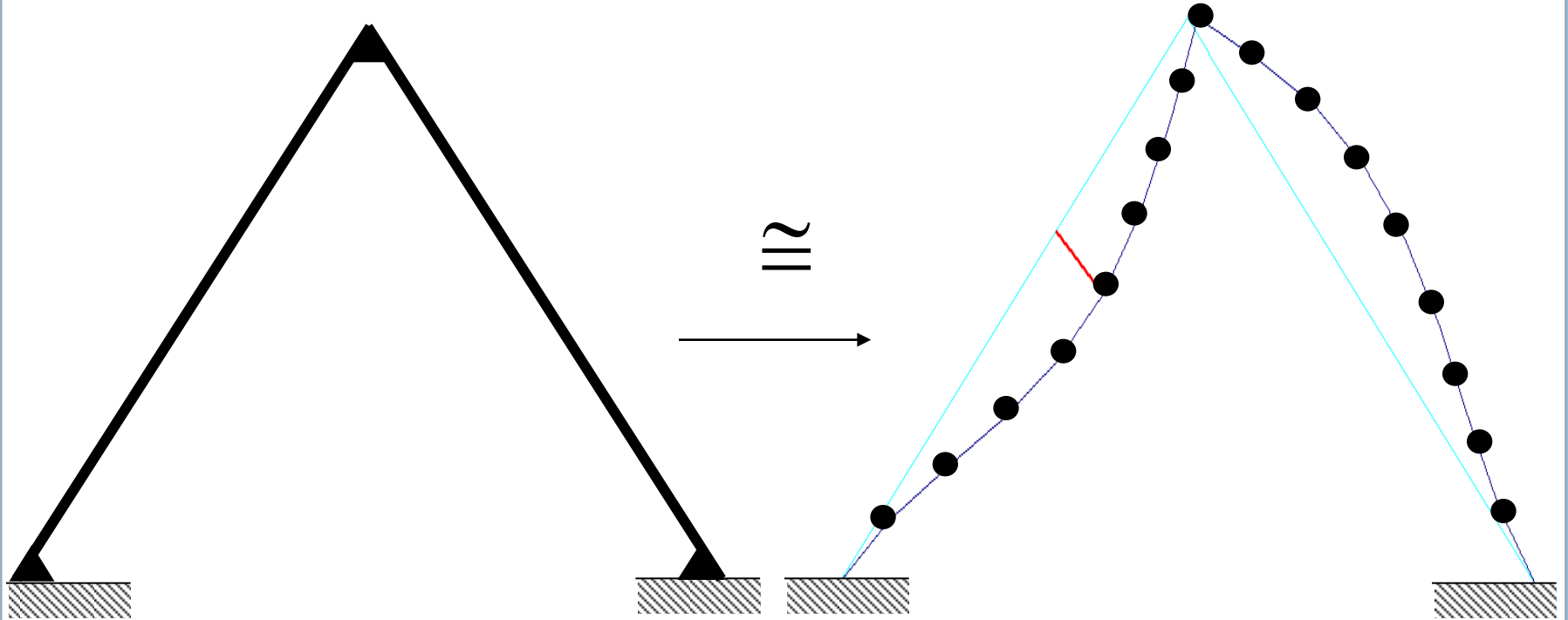
5. Real structural behavior in dynamics

The static „pin joint“ assumption has been incorporated in truss dynamics.

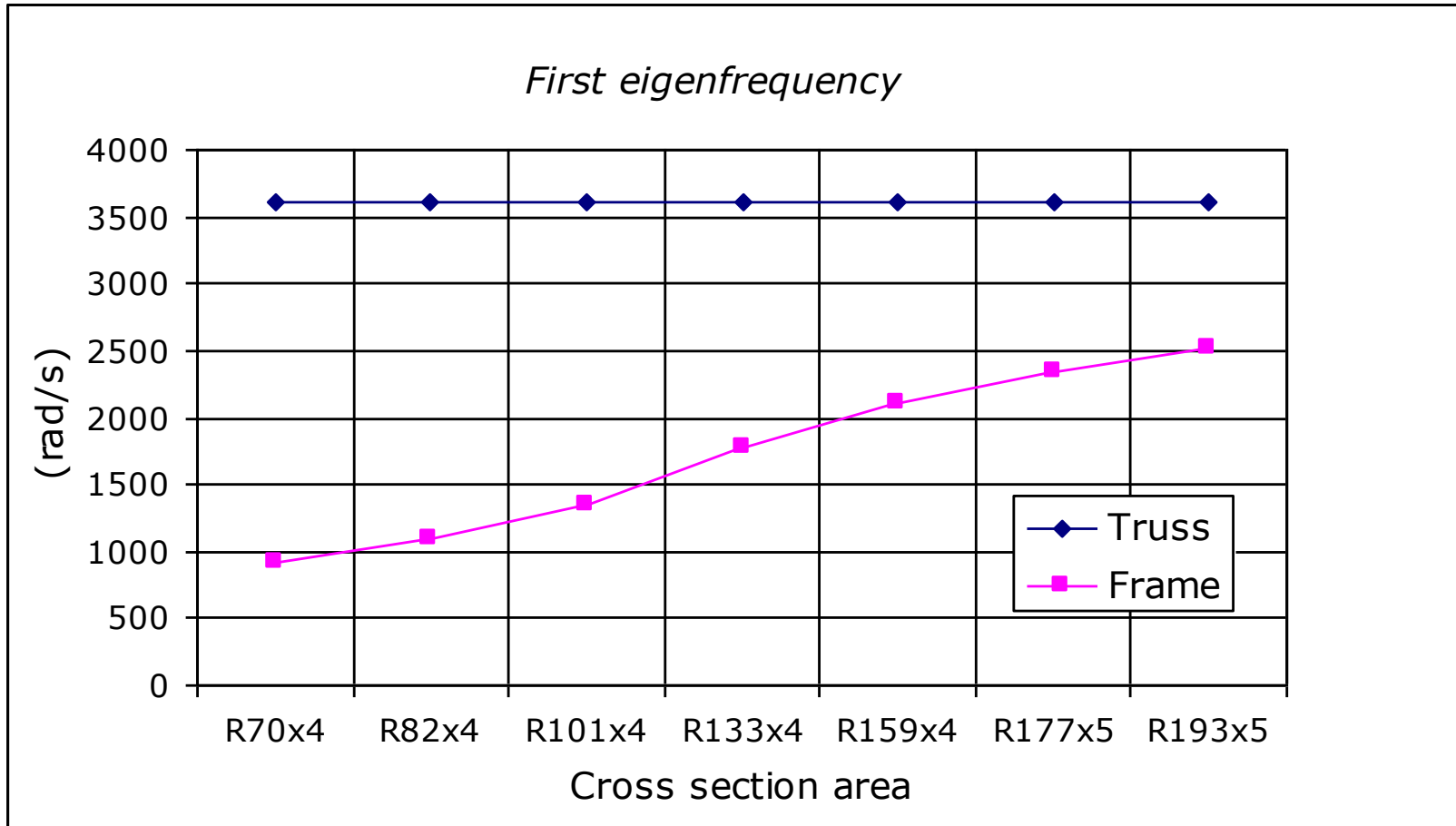
Equivalent masses of beams are attached to pin joints. Problem is reduced to vibration of concentrated masses connected with massless springs.



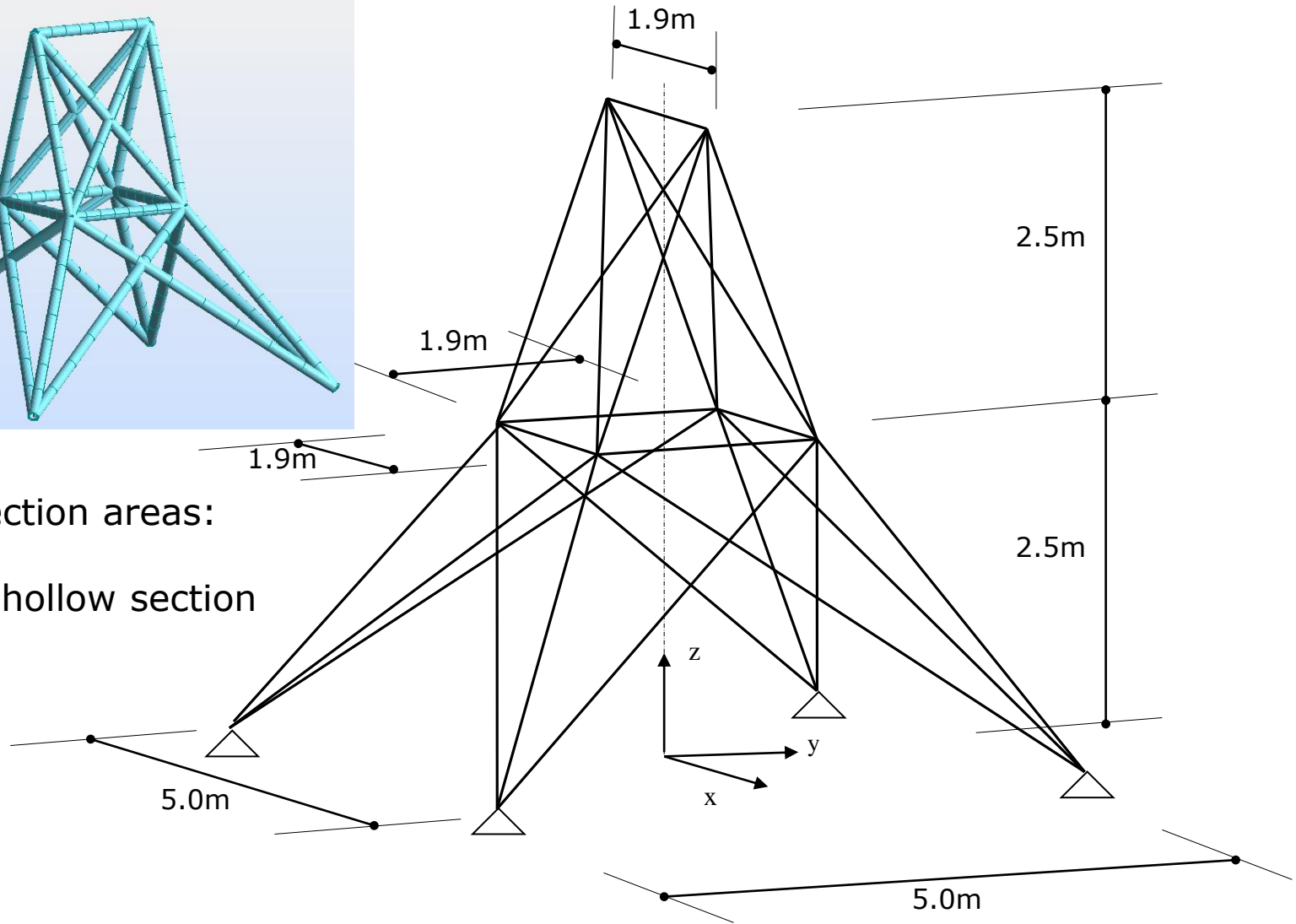
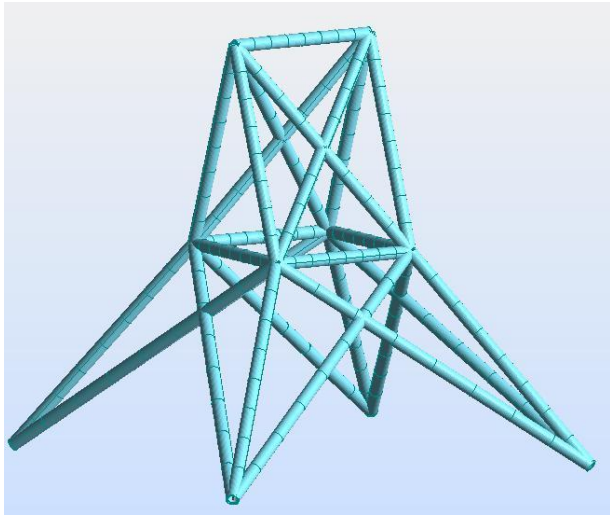
In real, rigidly connected structure, beam mass is distributed along its length and transversal motion assumed.



First eigenfrequency for both cases



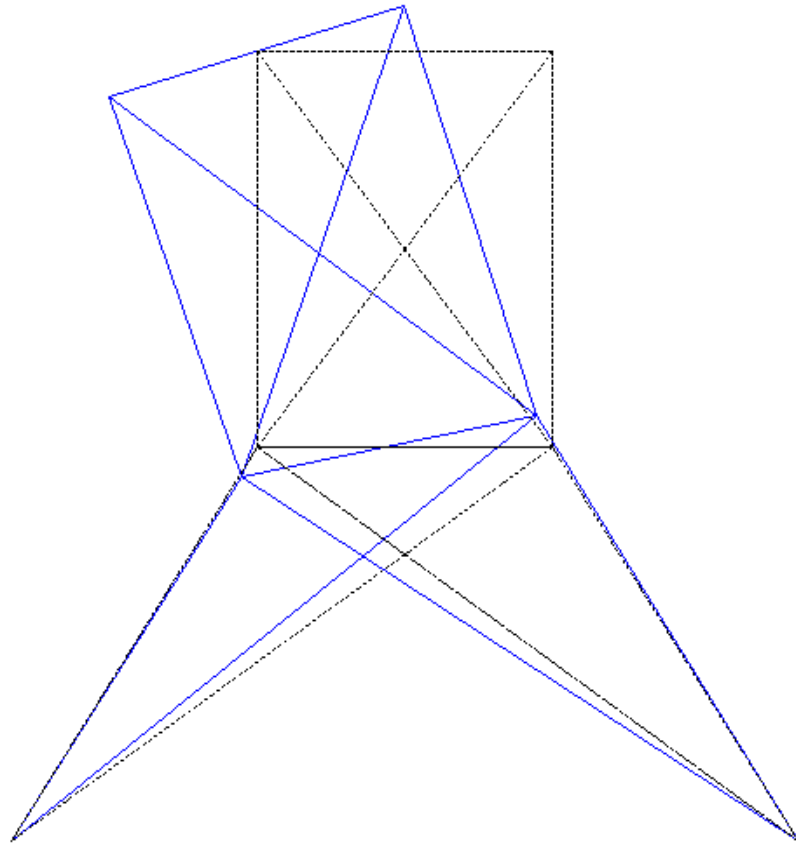
25 bar structure



Cross section areas:

Circular hollow section
Ø159x8

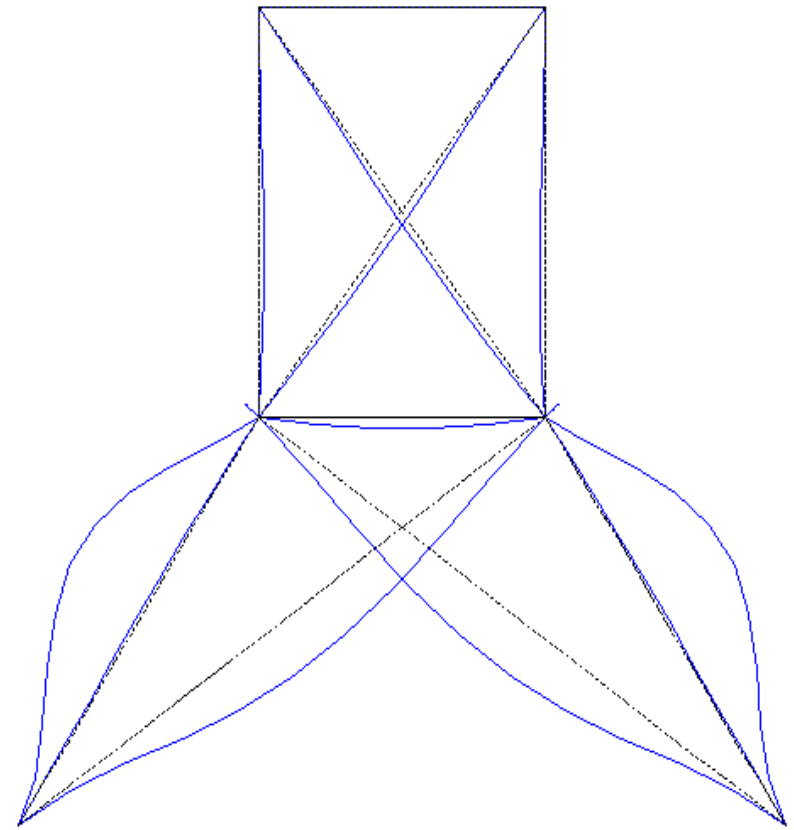
First mode shape of 25 bar structure



1ST FREQUENCY = 70.56 Hz FOR TRUSS

I – pin-joint model

f=70.56 Hz

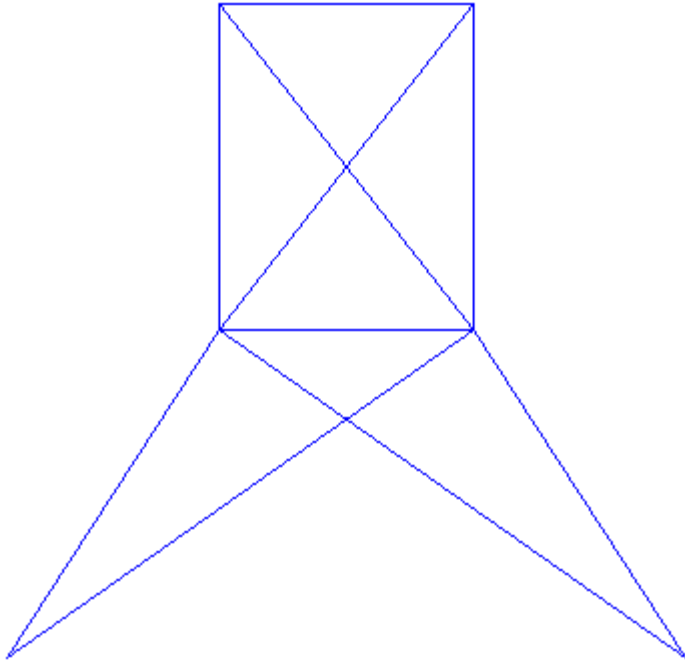


1ST FREQUENCY = 38.43 Hz FOR FRAME

II – rigid-joint model

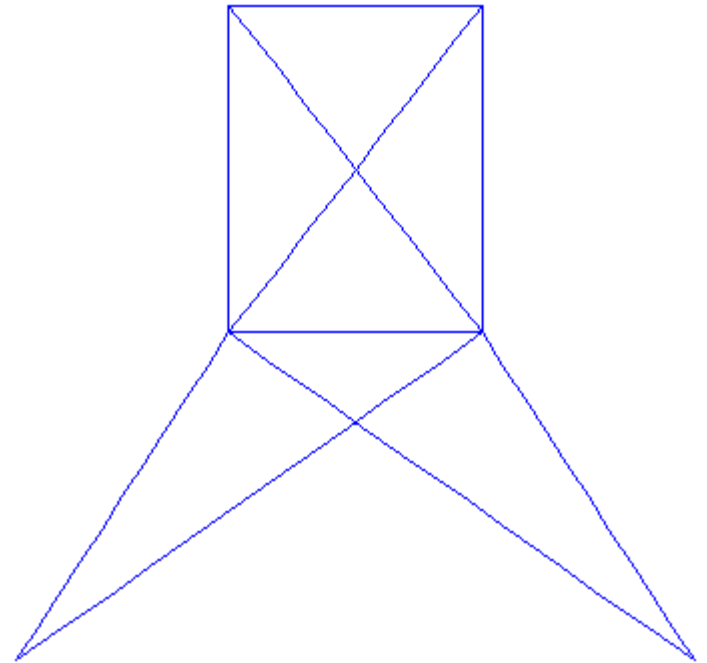
f=38.43 Hz

First mode shape animation



I – pin-joint model

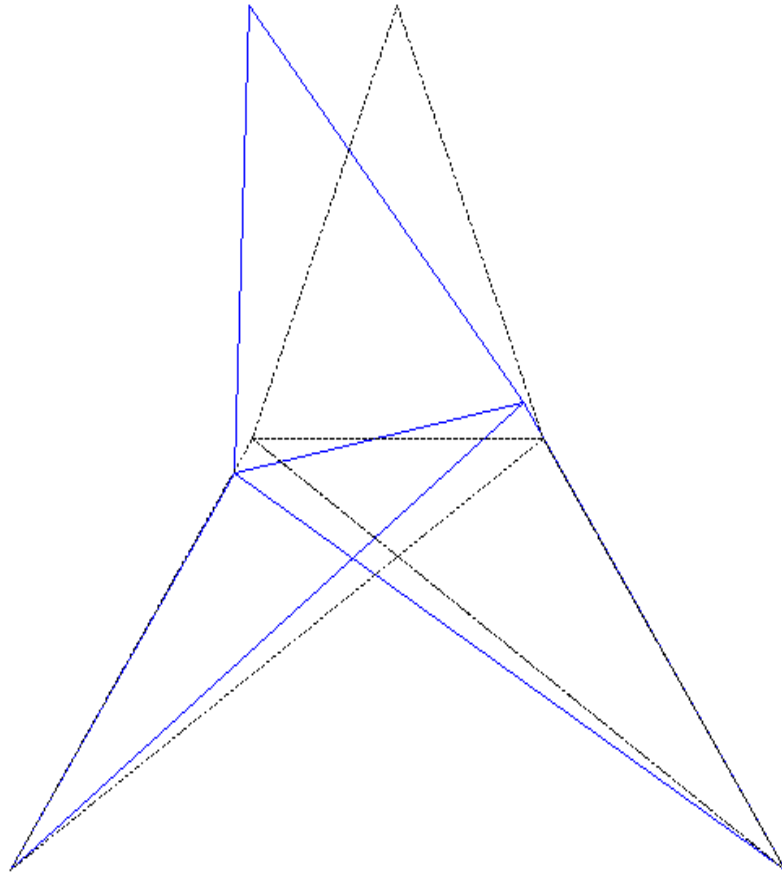
$f=70.56$ Hz



II – rigid-joint model

$f=38.43$ Hz

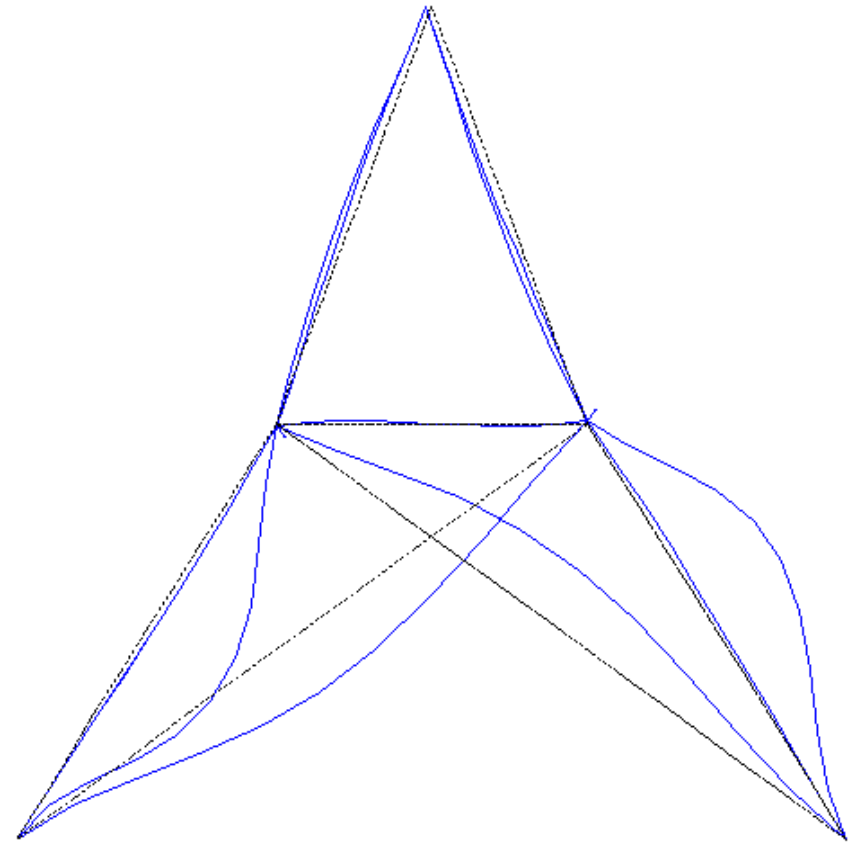
Second mode shape of 25 bar structure



2ND FREQUENCY = 73.63 Hz FOR TRUSS

I – pin-joint model

f=73.63 Hz



2ND FREQUENCY = 39.90 Hz FOR FRAME

II – rigid-joint model

f=39.90 Hz

Truss tower

Cross section areas:

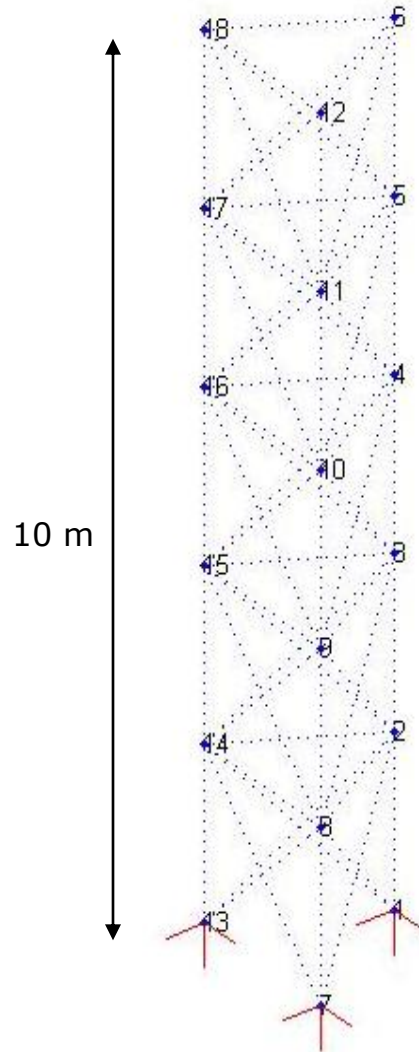
CHS Ø159/8

Material:

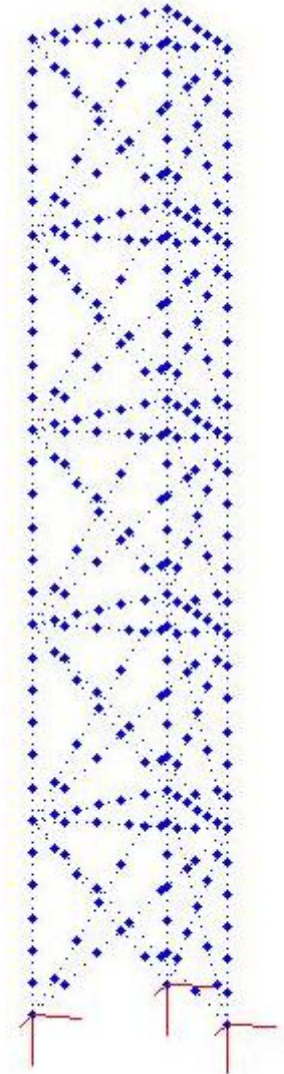
Steel

$E_x = 205 \cdot 10^6 \text{ kPa}$

$\rho = 7850 \text{ kg/m}^3$



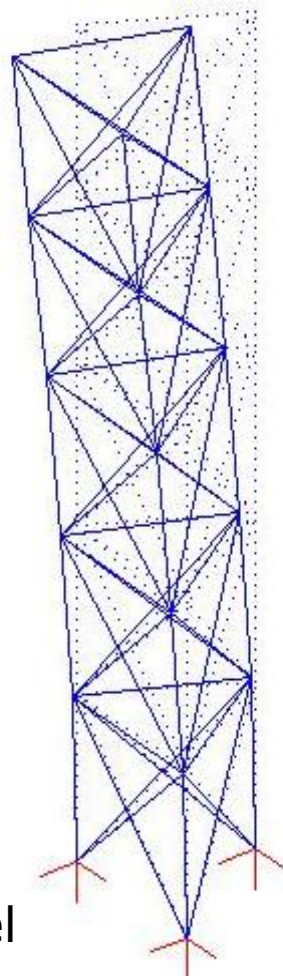
I pin-joint model



II rigid-joint model

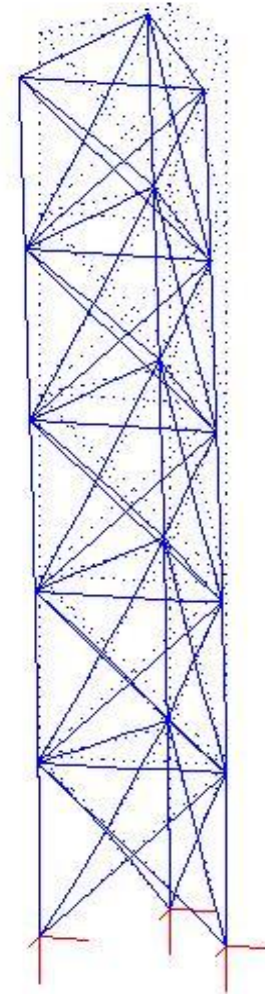
First mode shape of tower truss

$$\omega_1 = 64.4 \text{ rad/s}, f_1 = 10.3 \text{ Hz}$$



Pin-joint model
1st bending mode

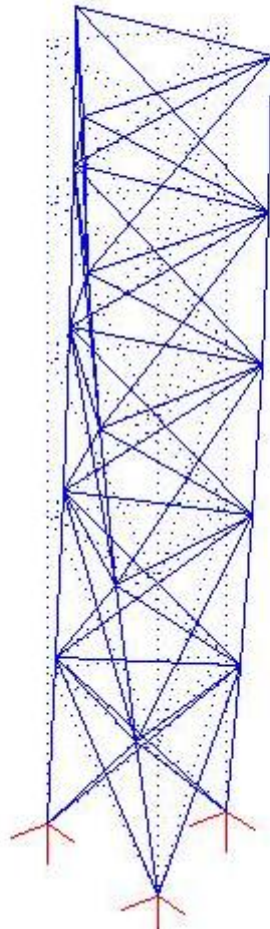
$$\omega_1 = 65.3 \text{ rad/s}, f_1 = 10.4 \text{ Hz}$$



Rigid-joint model
1st bending mode

Third mode shape of truss tower

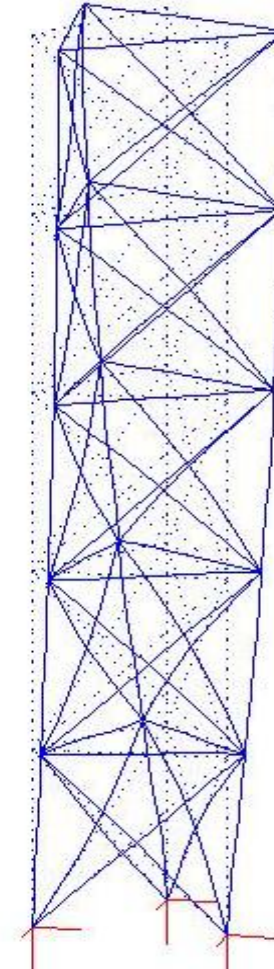
$$\omega_3 = 180 \text{ rad/s}, f_3 = 28.7 \text{ Hz}$$



Pin-joint model

1st torsional mode

$$\omega_3 = 198 \text{ rad/s}, f_3 = 31.6 \text{ Hz}$$

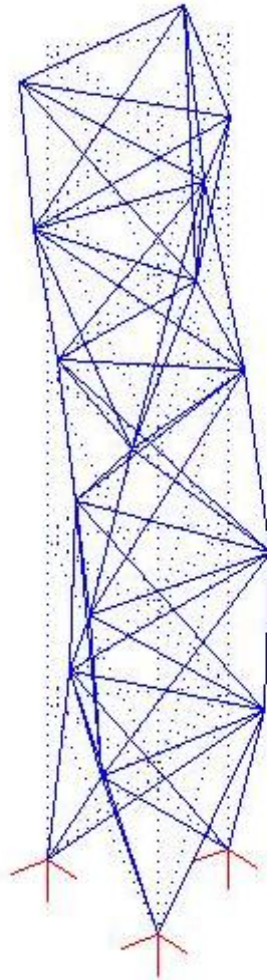


Rigid-joint model

1st torsional mode

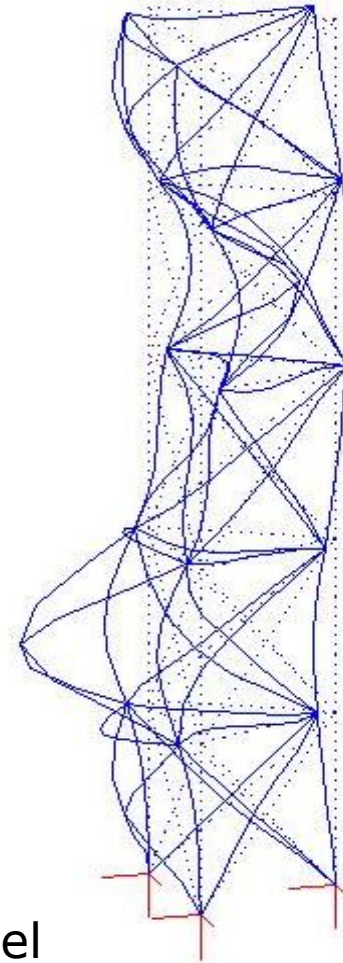
7th mode shape of truss tower

$$\omega_7 = 521 \text{ rad/s}, f_7 = 82.9 \text{ Hz}$$



Pin-joint model
2nd torsional mode

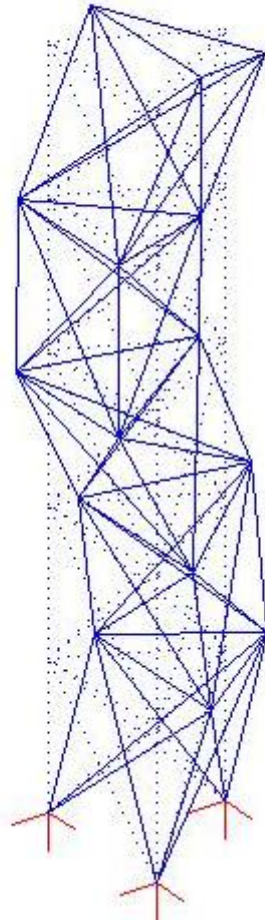
$$\omega_7 = 521 \text{ rad/s}, f_7 = 82.9 \text{ Hz}$$



Rigid-joint model
Local bending mode of bracings

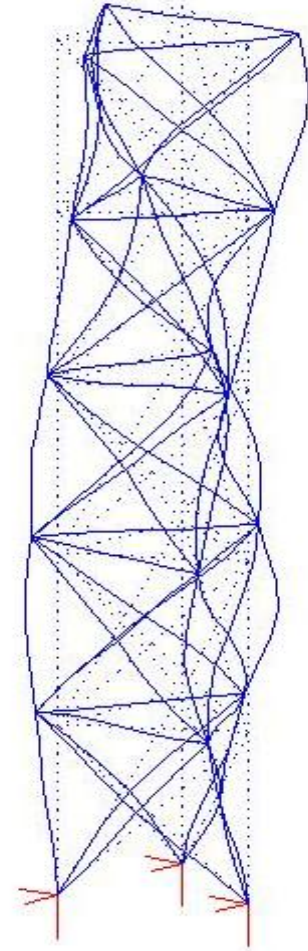
9th mode shape of truss tower

$$\omega_g = 731 \text{ rad/s}, f_g = 116 \text{ Hz}$$



Pin-joint model
2nd bending mode

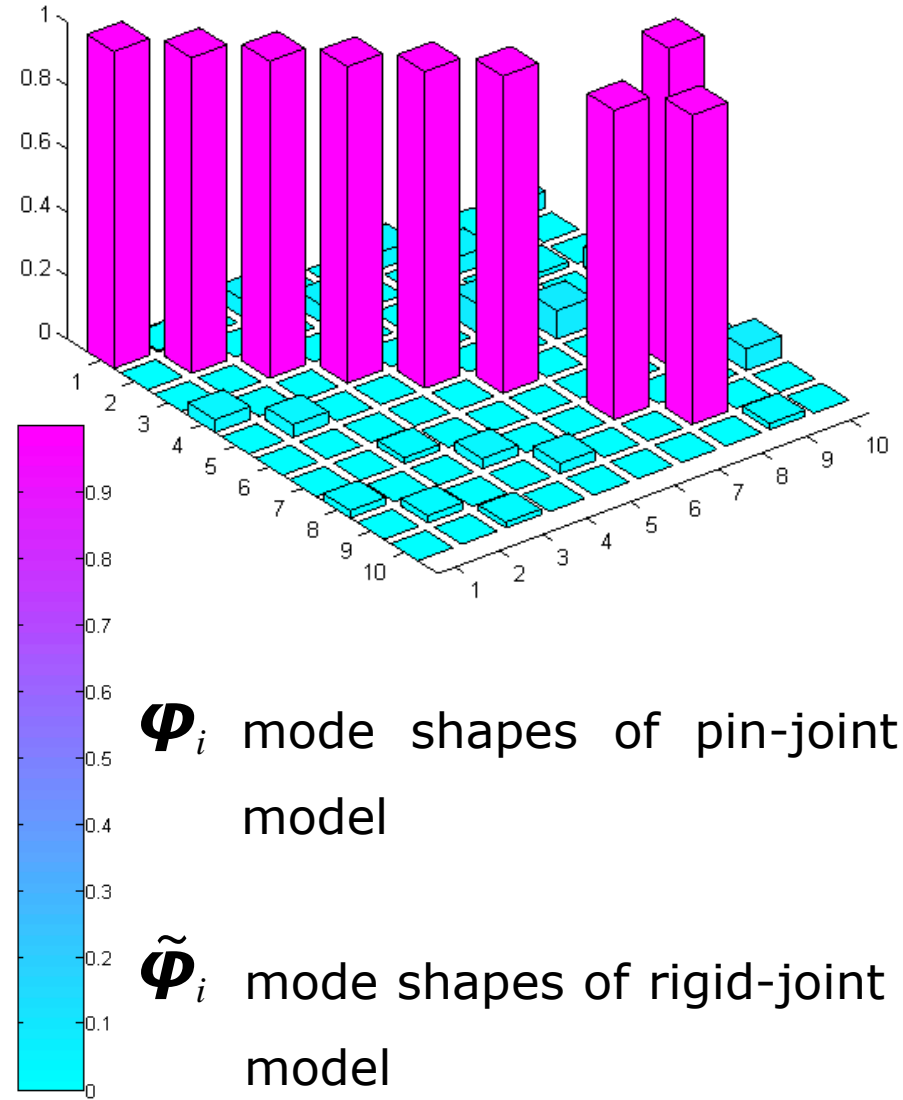
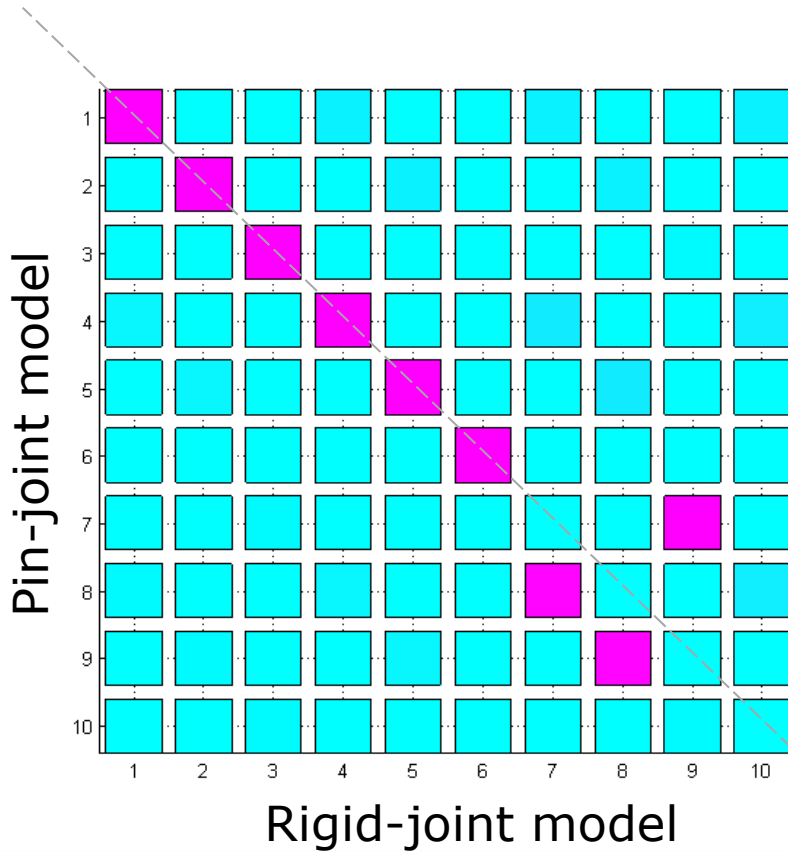
$$\omega_g = 570 \text{ rad/s}, f_g = 90.7 \text{ Hz}$$



Rigid-joint model
2nd torsional mode

Correlation of modes

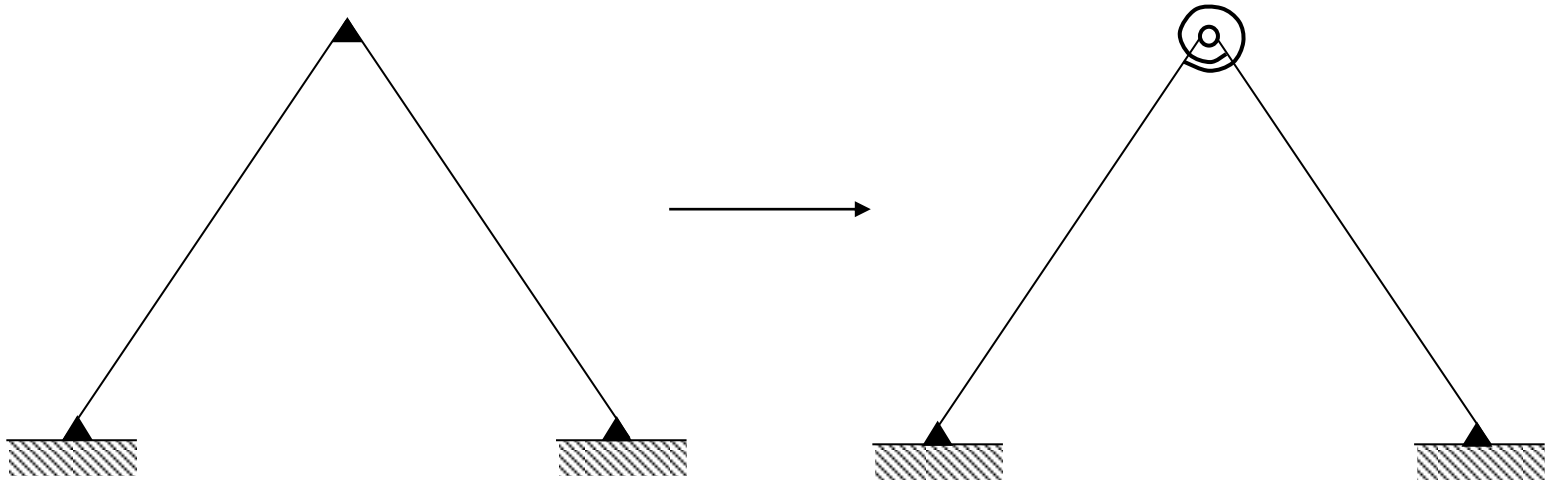
$$\text{MAC}_{ij} = \frac{(\boldsymbol{\varphi}_i^T \tilde{\boldsymbol{\varphi}}_j)^2}{(\boldsymbol{\varphi}_i^T \boldsymbol{\varphi}_i)(\tilde{\boldsymbol{\varphi}}_j^T \tilde{\boldsymbol{\varphi}}_j)}$$



$\boldsymbol{\varphi}_i$ mode shapes of pin-joint model

$\tilde{\boldsymbol{\varphi}}_i$ mode shapes of rigid-joint model

Damaged joint modeling



$$\mathbf{M}\ddot{\mathbf{q}} + \mathbf{K}\mathbf{q} = \mathbf{0}$$
$$(\mathbf{K} - \lambda_i^2 \mathbf{M})\boldsymbol{\varphi}_i = \mathbf{0}$$

$$i = 1, 2, \dots, N_m$$

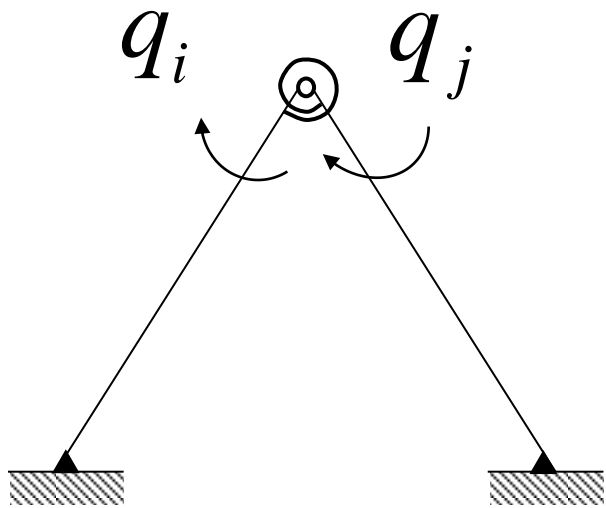
$$\mathbf{M}\ddot{\mathbf{r}} + (\mathbf{K} + \Delta\mathbf{K})\mathbf{r} = \mathbf{0}$$
$$(\mathbf{K} + \Delta\mathbf{K} - \omega_l^2 \mathbf{M})\boldsymbol{\psi}_l = \mathbf{0}$$

$$l = 1, 2, \dots, N_f$$

$$\Delta\mathbf{K} = \sum_{j=1}^{N_k} k_j \mathbf{B}_j$$

Stiffness correction

Damaged joint modeling continued



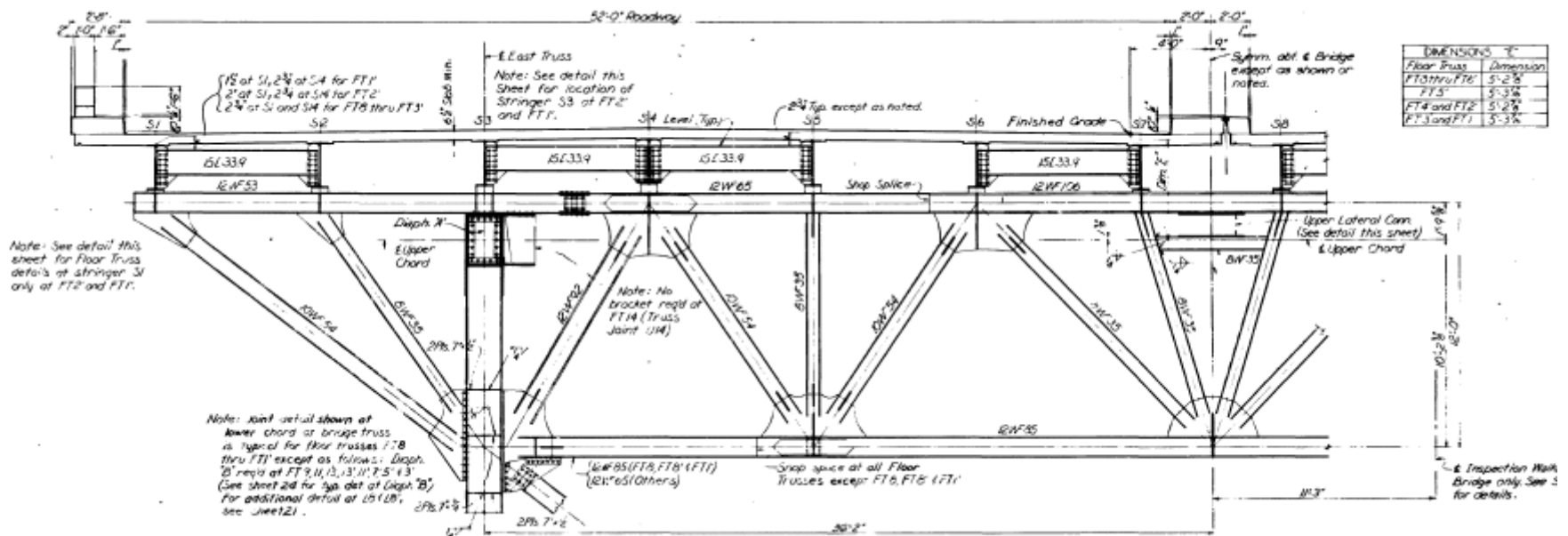
$$\mathbf{B}_j = \begin{bmatrix}
 0 & 0 & \dots & & 0 & 0 \\
 0 & \ddots & & & \ddots & 0 \\
 & & & 1 & 0 & -1 \\
 \vdots & & & 0 & 0 & 0 \\
 & & & -1 & 0 & 1 \\
 0 & \ddots & & & \ddots & 0 \\
 0 & 0 & \dots & & 0 & 0
 \end{bmatrix}$$

1	2	...	i	...	j	...	n_{DOFs}
							1
							2
							\vdots
							i
							\vdots
							j
							\vdots
							n_{DOFs}

I-35W truss bridge



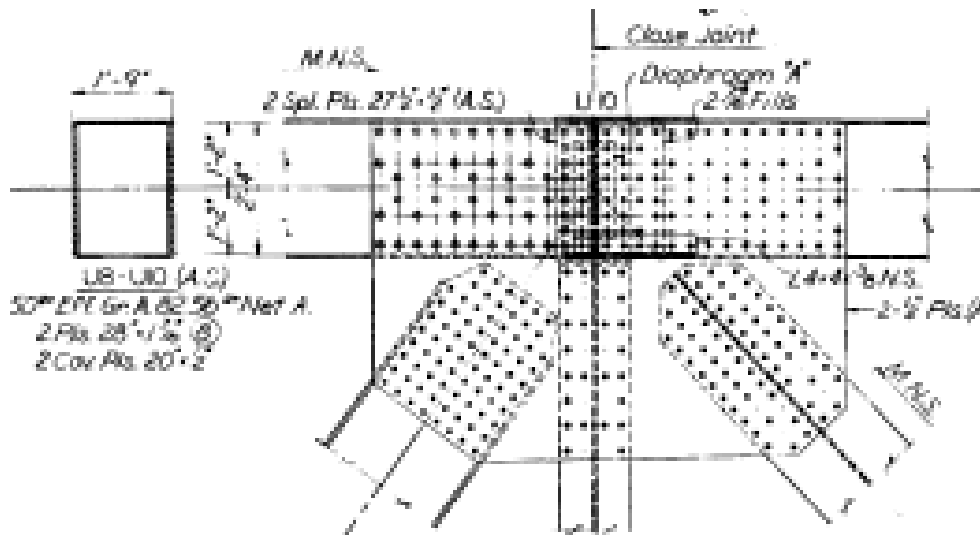
Floor truss according to construction plans



*) Astaneh-Asl, A., "Progressive Collapse of Steel Truss Bridges, the Case of I-35W Collapse", 7th International Conference on Steel Bridges, Guimarães, Portugal, 4-6 June, 2008.

Collapse

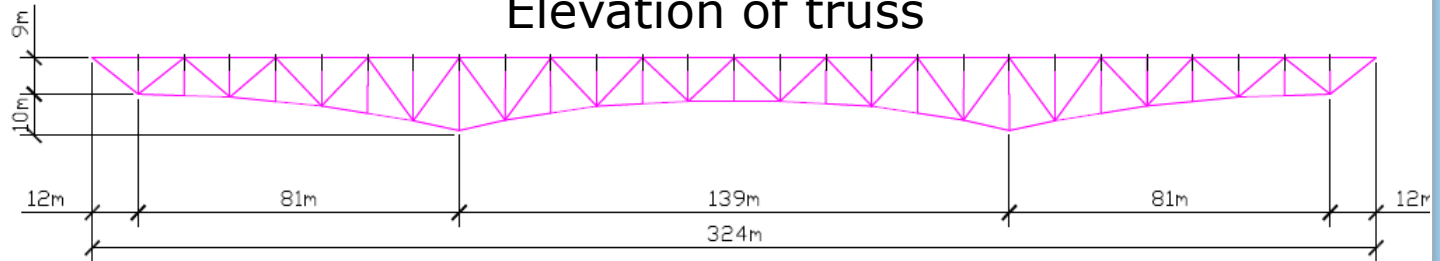
On August 1, 2007, the 40 years old I-35W steel deck truss bridge in Minneapolis, suddenly and without almost any noticeable warning collapsed.



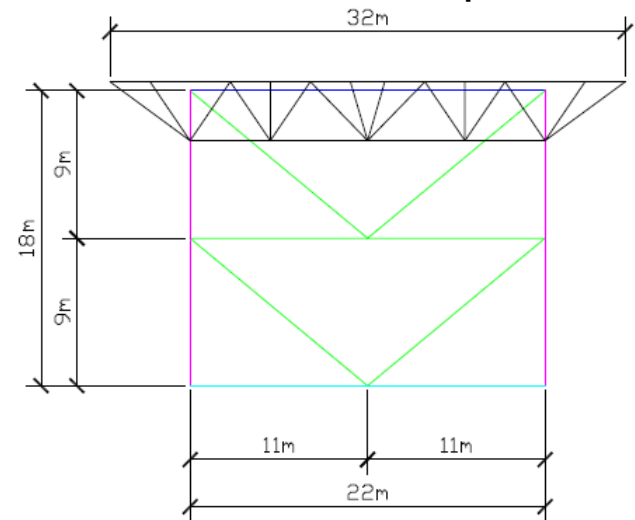
Joint detail

Computational model

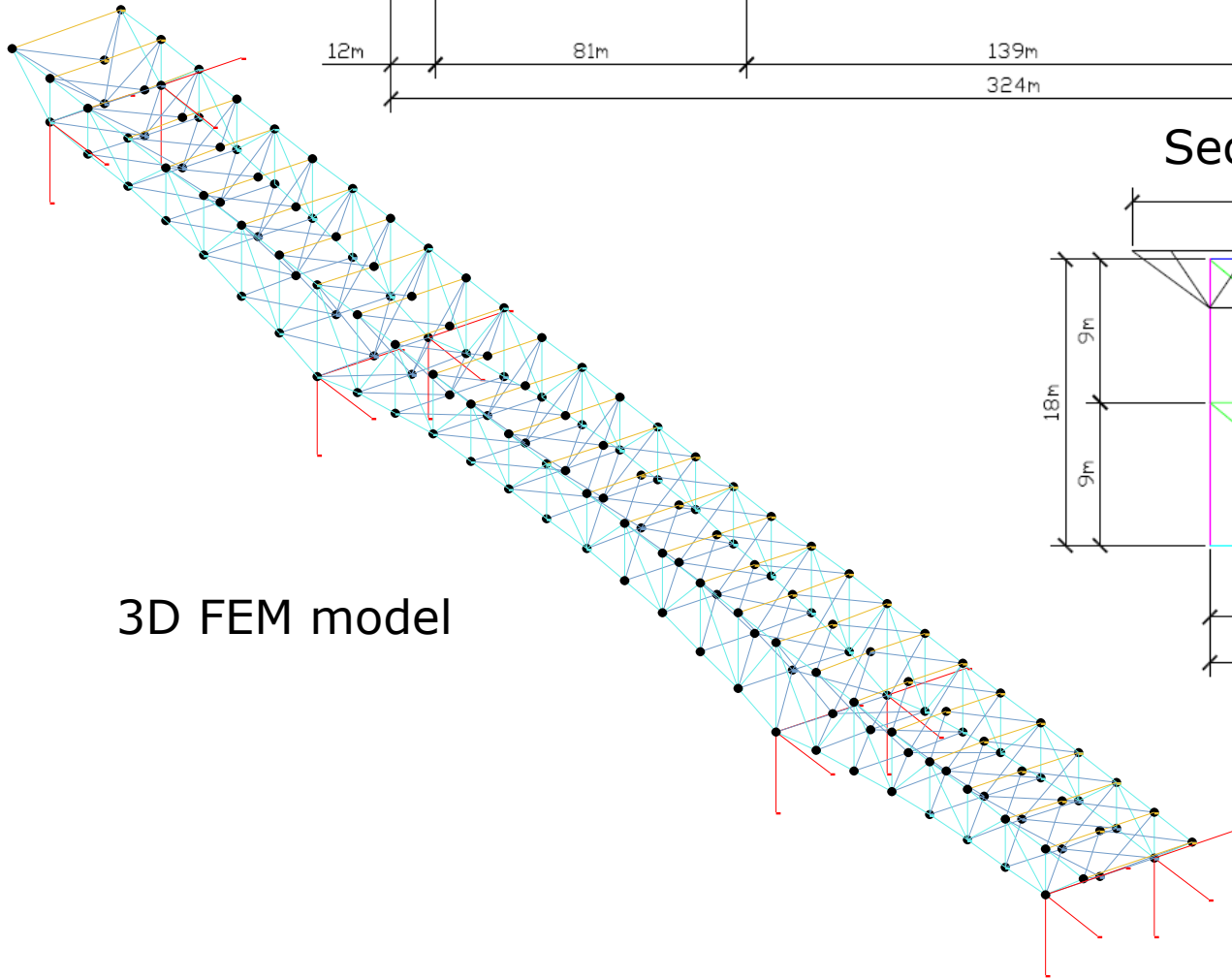
Elevation of truss



Section near pier 7

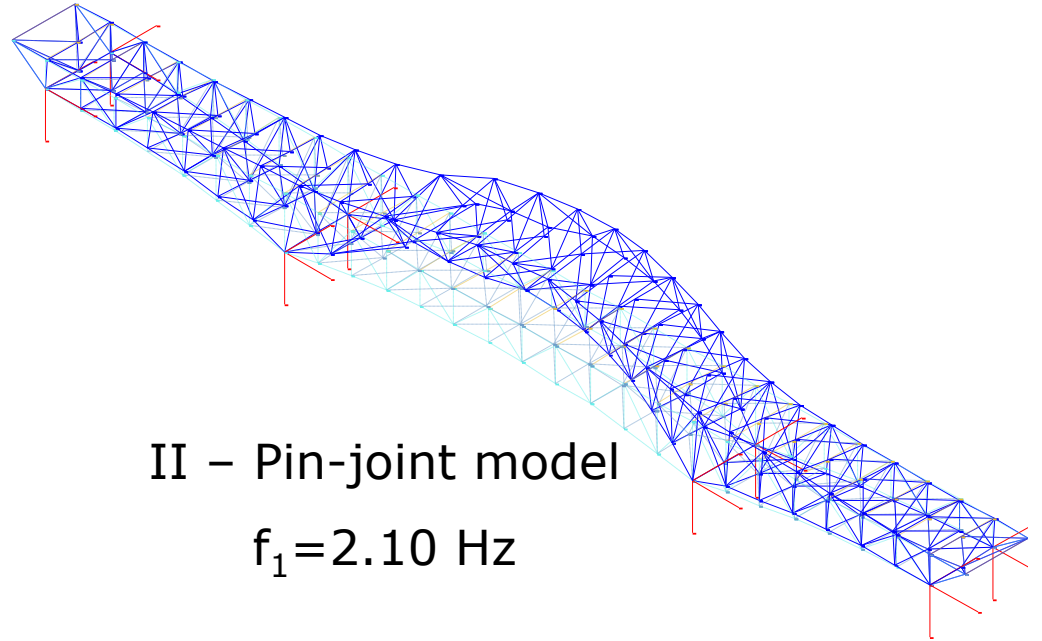


3D FEM model



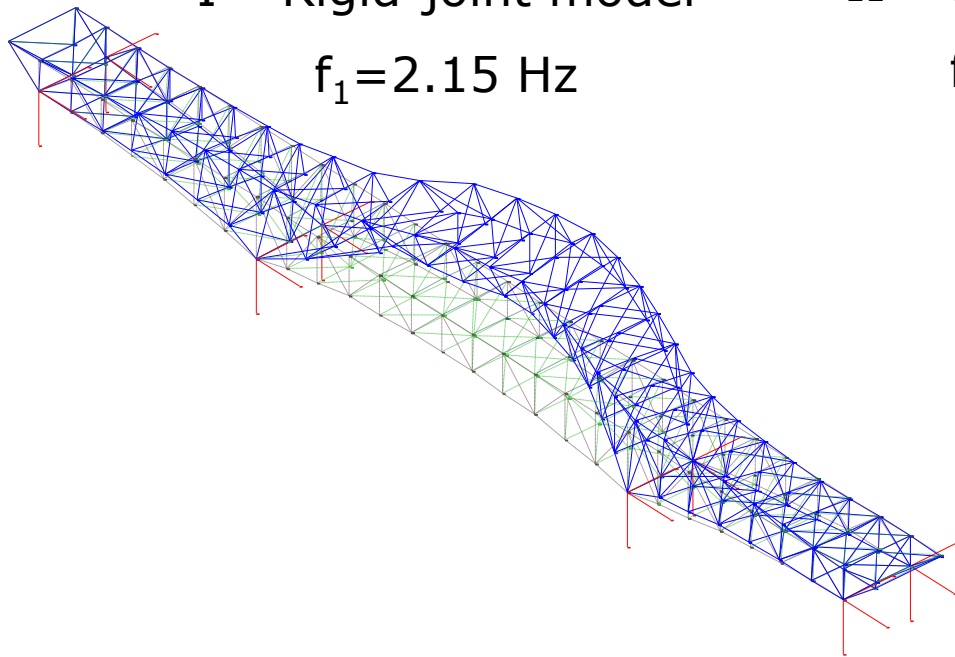
- $A_1 = 0.01258 \text{ m}^2$
- $A_2 = 0.017632 \text{ m}^2$
- $A_3 = 0.042101 \text{ m}^2$

1st mode – out-of-plane bending



II – Pin-joint model

$f_1=2.10$ Hz



I – Rigid-joint model

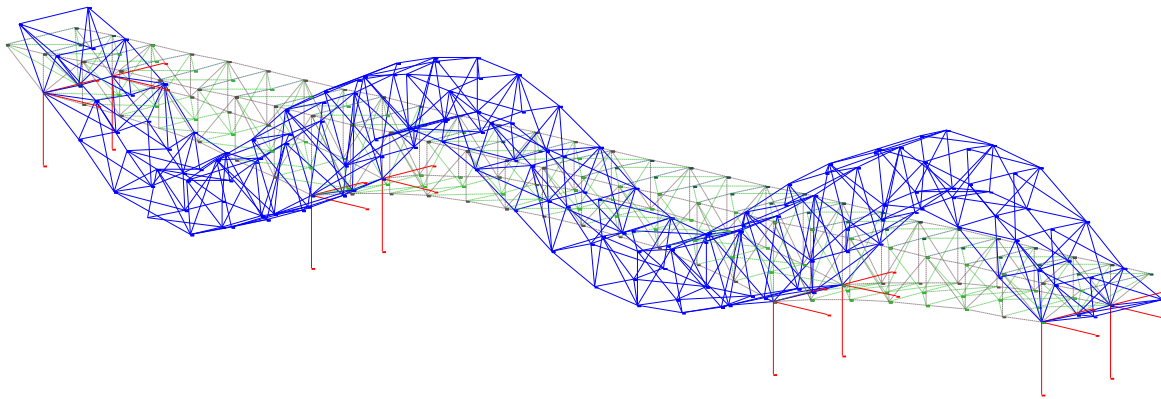
$f_1=2.15$ Hz

6th mode shape

I – Rigid-joint model

$$f_6 = 4.34 \text{ Hz}$$

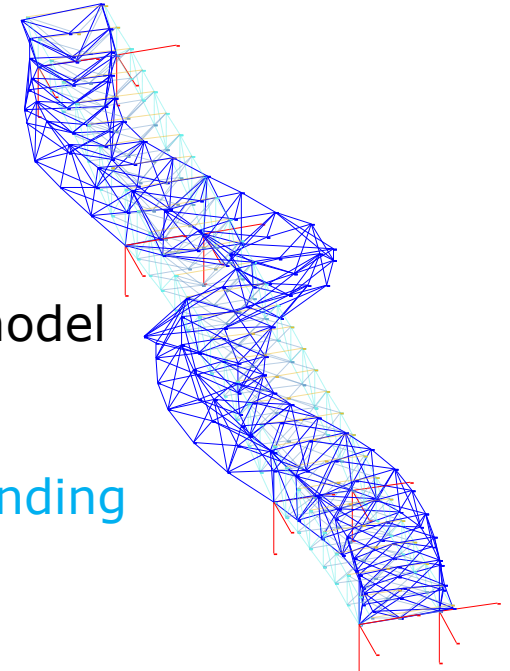
In-plane bending



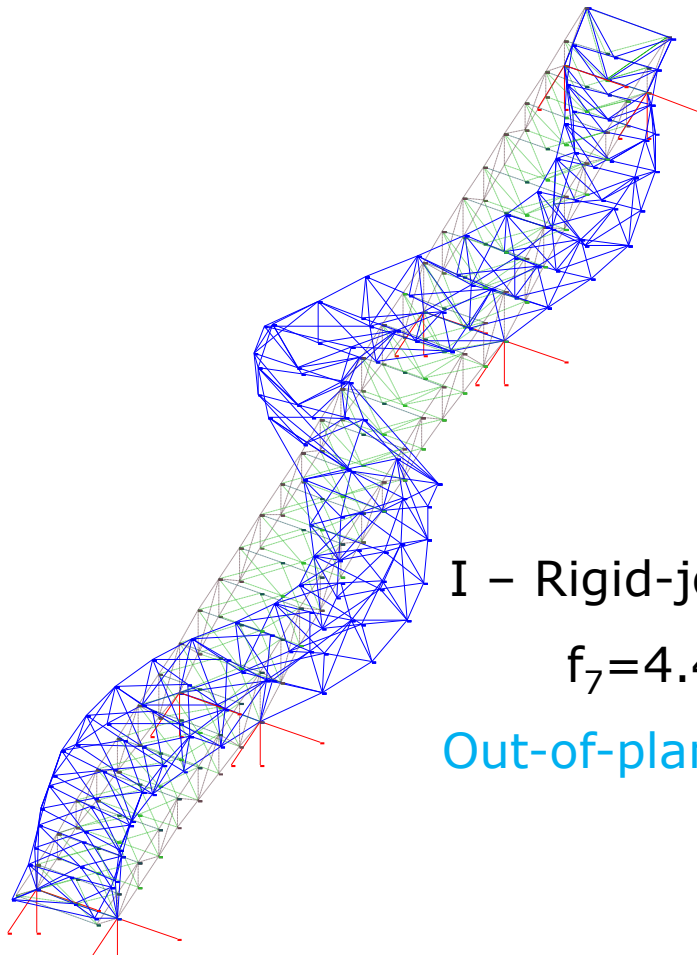
II – Pin-joint model

$$f_6 = 4.44 \text{ Hz}$$

Out-of-plane bending



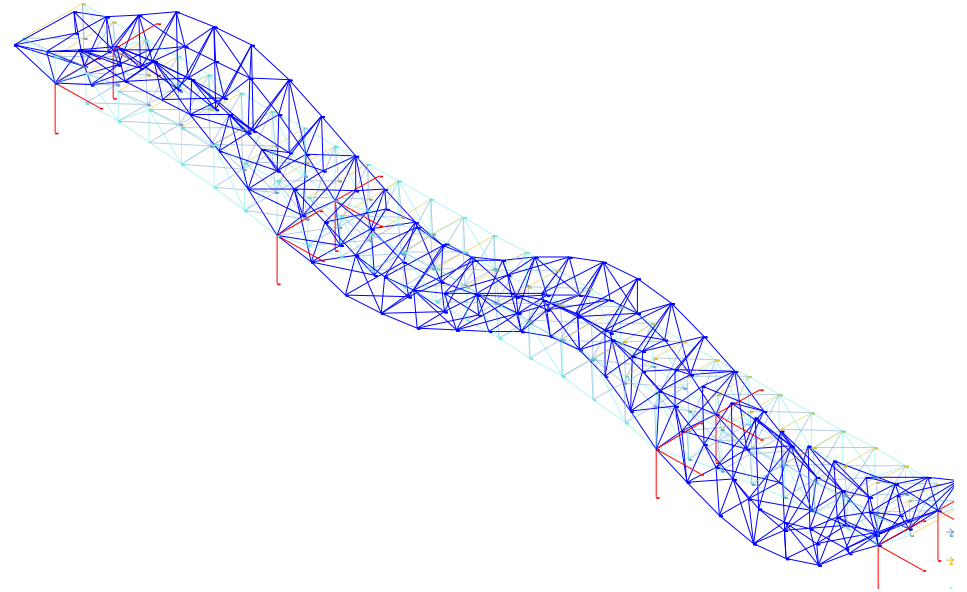
7th mode shape



I – Rigid-joint model

$f_7=4.48$ Hz

Out-of-plane bending



II – Pin-joint model

$f_7=4.49$ Hz

In-plane bending

7. Conclusions

- ❑ Theoretical background, together with presented examples show, that the pin-joint assumption in truss dynamics can lead to considerable errors in finding eigenfrequencies and eigenmodes.
- ❑ The pin-joint assumption doesn't allow to monitor, real structural joints, which are more often subjected to damages than prismatic rolled structural elements.

Acknowledgments

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Thank you for your attention.